

**Duration: 3h + additional 2x15 min** You should return scanned hand written answers as in a contact exam in a good pdf-quality

**It is compulsory to solve only THREE (3) EXERCISES that you choose freely: only three best exercises (answers) will be graded even if the student solves four.**

- 1) Results given without shown the logical steps needed to achieve them will be ignored even if correct.
- 2) Sequentially number (numeroi juoksevasti) your answer papers 1(n) ... n(n), where n = total number of pages
- 3) Write readably your name, family name and student number.
- 4) Name the pdf-file: Student\_ID\_Name\_Date.pdf Make a good quality scanned pdf
- 5) All additional material, like listings, graphs can be appended as pdfs to the answer
- 6) Please check the physical units of your answers.

Exam. points	Grade
≥ 13,75	5
12,75	4
9,75	3
8,25	2
7	1
< 7	fail=0

### Examination 26/10/2021

The material is linear elastic in all the structures below

1. Use the dummy unit-load theorem (or method) and determine the horizontal displacement at roller C [3 point]. Account for both the effects of bending and axial forces when computing displacement. Ignore the shear effects. [overall = 5 points]

$$EI = \alpha EA \cdot l^2$$

Hints: is it statically determined? Determine support reactions Then determine and draw accurately the bending moment [1 point] and axial force [1 point] diagrams

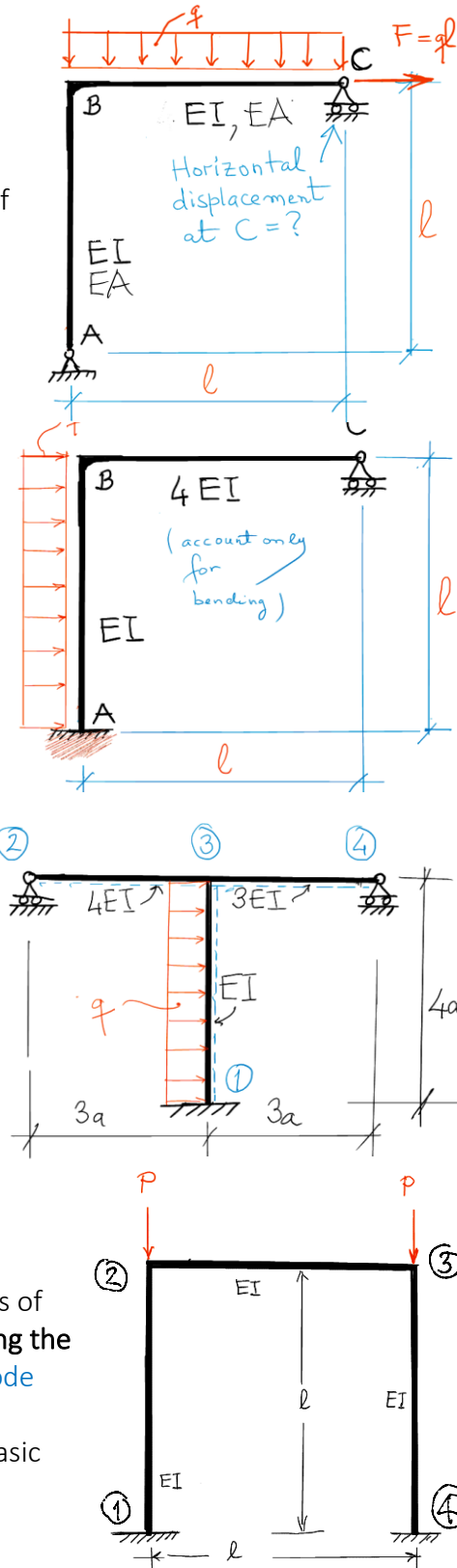
2. Use the general force method and
  - a) determine the bending moment at support A [3 points] and draw accurately the bending moment diagram [1 point]. Account only for effects of bending
  - b) Determine the support reaction at C (value and direction). [1 p].

3. Use Slope-Deflection Method and
  - a) determine the bending moment at clamping support 1 [4 points]
  - b) use results from question a) and determine the horizontal displacement at roller 4 [1 point] (all other methods are welcomed for evaluating the displacement)

Hint a): If you wish you can use the stiffness-moment relation for hinged beams where appropriate

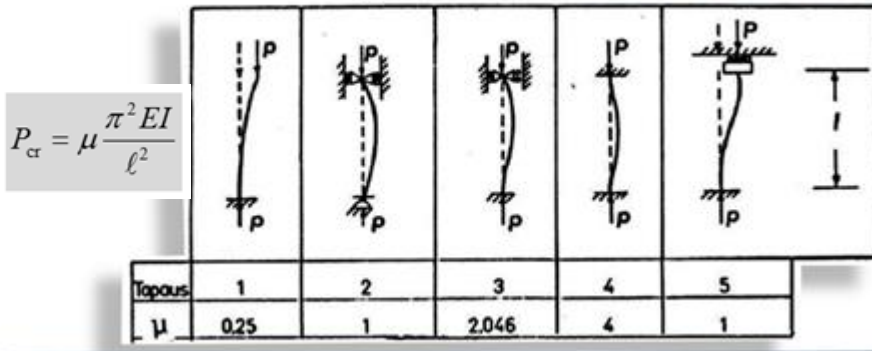
### 4. Buckling of sway-frames

- The Frame is loaded symmetrically with by two concentrated loads P. Use Slope-Deflection Method and 1) derive the explicit expression, in terms of Berry's stability functions, of the needed **criticality condition for determining the critical buckling load P** [3 points]. Hint: assume **anti-symmetric buckling mode**
- 2) solve numerically for the value of the buckling load P [1 point].
  - 3) Give a **bracket** for the value of buckling load using cleverly the Euler's basic cases (see tables in the formulary) [1 point].



Euler's basic buckling cases

Eulerin perusnurjahduks



$$M_{ij} = A_{ij}\phi_{ij} + B_{ij}\phi_{ji} - C_{ij}\psi_{ij} + \bar{M}_{ij} \quad M_{ij}^0 = A_{ij}^0\phi_{ij} - C_{ij}^0\psi_{ij} + \bar{M}_{ij}^0$$

Beam-column with constant flexural rigidity:

$$A_{ij} = A_{ji} = \frac{2\psi(kL)}{4\psi^2(kL) - \phi^2(kL)} \frac{6EI}{L}, \quad B_{ij} = B_{ji} = \frac{\phi(kL)}{4\psi^2(kL) - \phi^2(kL)} \frac{6EI}{L}$$

$$C_{ij} = A_{ij} + B_{ij}, \quad A_{ij}^0 = C_{ij}^0 = \frac{1}{\psi(kL)} \frac{3EI}{L}$$

one end is hinged  $B_{ij} = B_{ji} = 0$

$$kL \equiv L \sqrt{\frac{P}{EI}}$$

Berry's functions:

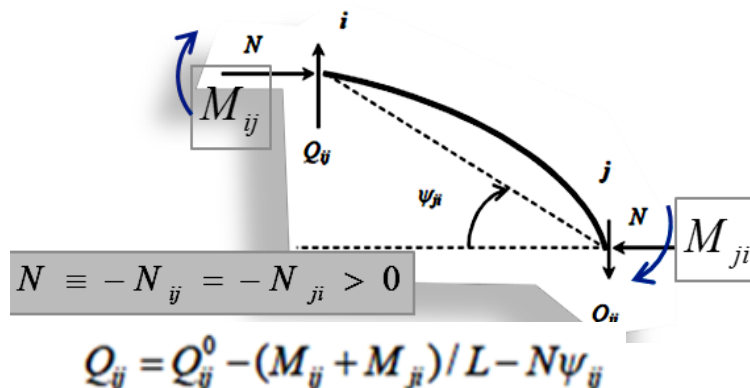
Olkoon  $\lambda \equiv kL$ ,  $\lambda \equiv kL$

Puristettu sauva:

Compression:  $\phi(\lambda) = \frac{6}{\lambda} \left( \frac{1}{\sin \lambda} - \frac{1}{\lambda} \right)$ ,  $\psi(\lambda) = \frac{3}{\lambda} \left( \frac{1}{\lambda} - \frac{1}{\tan \lambda} \right)$ , ja  $\chi(\lambda) = \frac{24}{\lambda^3} \left( \tan \frac{\lambda}{2} - \frac{\lambda}{2} \right)$ .

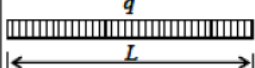
Vedetty sauva:

Extension:  $\phi(\lambda) = \frac{6}{\lambda} \left( -\frac{1}{\sinh \lambda} + \frac{1}{\lambda} \right)$ ,  $\psi(\lambda) = \frac{3}{\lambda} \left( -\frac{1}{\lambda} + \frac{1}{\tanh \lambda} \right)$ , ja  $\chi(\lambda) = \frac{24}{\lambda^3} \left( -\tanh \frac{\lambda}{2} + \frac{\lambda}{2} \right)$ .



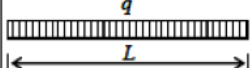
$$\bar{M}_{12} \equiv MK_1$$

$$\bar{M}_{ij}, \bar{M}_{ji}$$

N:o	Kuormitus	Kiinnitysmomentit:
1		$MK_1 = -\frac{qL^2}{12}, MK_2 = \frac{qL^2}{12}$
2	...	...

$$\bar{M}_{ij}$$

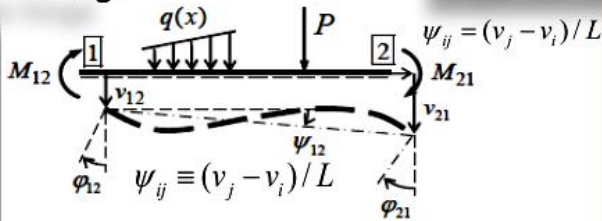
$$\bar{M}_{ji}$$

N:o	Kuormitus	Nivel oikeassa päässä:	Nivel vasemmassa päässä:
1		$MK_1^0 = -\frac{qL^2}{8}$	$MK_2^0 = \frac{qL^2}{8}$

The stiffness equation relating the end-moments to the end-displacements

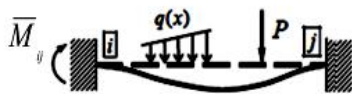
If you are using lecture's notations

No hinge



$$M_{ij} = a_{ij}\phi_{ij} + b_{ij}\phi_{ji} - c_{ij}\psi_{ij} + \bar{M}_{ij}, \quad i \neq j$$

$$a_{ij} = \frac{4EI}{L}, \quad b_{ij} = \frac{2EI}{L}, \quad c_{ij} = \frac{6EI}{L} \quad (EI\text{-constant})$$



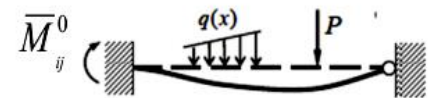
Fixed end-moment resulting from external mechanical loading, look from tables

One node is hinged

The is a superscript "0" means that the support at end j is hinged

$$M_{ij}^0 = a_{ij}^0\phi_{ij} - c_{ij}^0\psi_{ij} + \bar{M}_{ij}^0$$

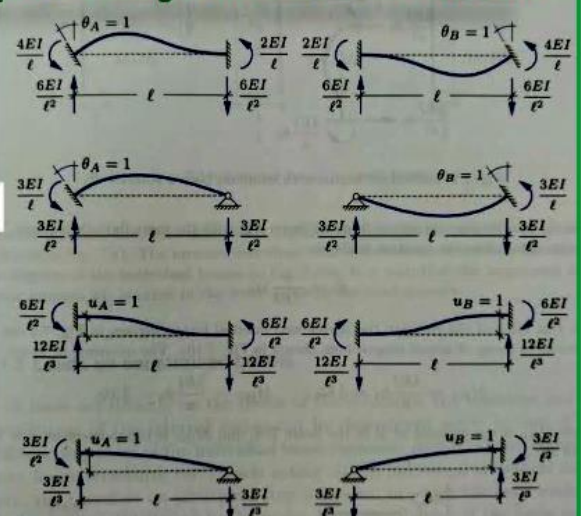
$$a_{12}^0 = c_{12}^0 = \frac{3EI}{L} \quad \psi_{ij} = (v_j - v_i) / L$$



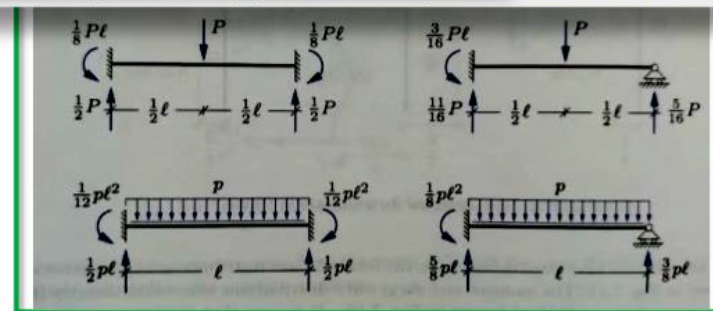
Fixed end-moment resulting from external mechanical loading, look from tables

If you are using Krenk's textbook notations

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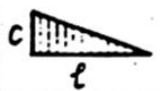


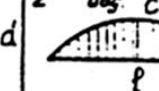
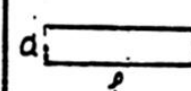
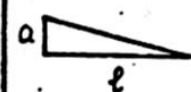
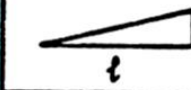
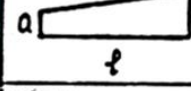
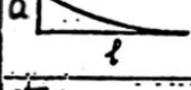
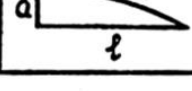


Bending Moments (pay attention to the sign convention to convert to Fixed-End-Moments)



# Maxwell-Mohr integrals table

TABEAU DES INTEGRALES  $\int_0^l i M^k M dx$

$k \backslash i$ $M$					
	$acl$	$\frac{1}{2}acl$	$\frac{1}{2}adl$	$\frac{1}{2}al(c+d)$	$\frac{2}{3}acl$
	$\frac{1}{2}acl$	$\frac{1}{3}acl$	$\frac{1}{6}adl$	$\frac{1}{6}al(2c+d)$	$\frac{1}{3}acl$
	$\frac{1}{2}bcl$	$\frac{1}{6}bcl$	$\frac{1}{3}bdl$	$\frac{1}{6}bl(c+2d)$	$\frac{1}{3}bcl$
	$\frac{1}{2}(a+b)cl$	$\frac{1}{6}(2a+b)cl$	$\frac{1}{6}(a+2b)dl$	$\frac{l}{6}[a(2c+d) + b(c+2d)]$	$\frac{1}{3}(a+b)cl$
	$\frac{1}{3}acl$	$\frac{1}{4}acl$	$\frac{1}{12}adl$	$\frac{1}{12}al(3c+d)$	$\frac{1}{5}acl$
	$\frac{2}{3}acl$	$\frac{5}{12}acl$	$\frac{1}{4}adl$	$\frac{1}{12}al(5c+3d)$	$\frac{7}{15}acl$